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Adsorption Cycle and Its Hybrid with Multi-Effect Desalination

Muhammad Wakil Shahzad, Kyaw Thu, Ang Li, Azhar Bin Ismail and Kim Choon Ng

Abstract

Adsorption (AD) cycle is recently pioneered for cooling and desalination applications. For water treatment, the cycle can be used to treat highly concentrated feed water, ranging from seawater, ground water, and chemically laden waste water. This chapter presents a review of the recent development of AD cycle and its hybridization with known conventional cycles such as the MED and MSF. We begin by looking at the basic sorption theory for different adsorbent–adsorbate pairs, namely the silica gel–water and the zeolite–water pairs. Under the IUPAC categorization, there are six types of isotherm behavior that capture almost all types of adsorbent–adsorbate behaviors and many isotherm correlations have been developed to described their uptake patterns, namely the Henry, Langmuir, Toth, etc. We have recently developed a correlation that can universally capture all six types of isotherms of IUPAC and it requires only four regression coefficients.

We present also the basic AD cycle for seawater desalination as well as its hybridization with known conventional thermally driven cycles. We present the performances of the AD pilot which was powered by renewable solar thermal input. Owing to thermodynamic synergy between the thermally driven cycles, the AD cycle is combined with the robust multi-effect distillation cycle to improve the water production yields. The hybrid cycle is called the “MED+AD” or MEDAD in short. With hybridization, it allows the bottom-brine temperature of the MED to operate below ambient temperature, as low as 5°C, in contrast to the conventional MED which is limited by the ambient, resulting in a quantum increase of distillate production by two to three times. We demonstrate this efficiency improvement in a pilot comprising...
a three-stage MED and AD plant and the top-brine temperature is maintained at 70°C. Lastly, we present the concept of exergy to apportion the operating cost of fuel input in a cogeneration plant where both electricity and water are being produced, and this concept gives an operating cost of fuel input to be lower than the conventional enthalpic approach.

Keywords: Thermal Desalination, Adsorption Desalination (AD), Exergy Analysis, Multi-Effect Distillation (MED), Hybrid Cycles

1. Introduction

Fresh water is a precious entity which is needed for economic development of every sector of a country such as agriculture, industrial, and domestic sectors. The persistent quest of economic development and exponential population growth in many countries has intensified the water demand which is projected to grow annually at a rate of 3–4% [1–3]. Even though 70% of the earth is covered by water, most of the part is in nonpotable state due to its salinity, either in the form of brackish, waste, and seawater [3–7]. About 20% of the world population is living below the acute water poverty level of 500 m$^3$ per capita per year [8], risking their health due to poor water quality and substandard sanitation. The demand for fresh water in 2030 is expected to increase up to 6,900 billion cubic meters (Bm$^3$), as compared to total available water resources of 4,500 Bm$^3$ [9]. The fresh water supply and demand scenario is shown in Figure 1, and it is predicted that the sustainable natural water cycle of our earth cannot meet the projected future water demand.

Figure 1. Fresh water supply demand gap: current and future estimates
In 2012, installed desalinated water capacity was 72 Mm$^3$/day (mcmd) and it is estimated to increase to 98 mcmd by 2015 as reported in the literature [10]. Almost half of total desalination capacity is installed in the Gulf Cooperation Council (GCC) countries. In the 1960s and 1970s, many GCC countries relied heavily on the ground water supply for all sectors such as agriculture, industrial, and domestic sectors. However, the ground water resource in these countries has depleted rapidly over the decades due to excessive water extraction and insufficient aquifer recharge. Despite a higher desalination market share in GCC, the fresh water availability is dropping rapidly to below the acute water poverty level of 500 m$^3$ per capital per year, caused by an exponential population growth and higher GDP thrust.

GCC countries population, fresh water availability, and desalination scenario are shown in Figure 2, spanning from the early decades in 1950 to the future years up to 2025 [11–17]. Seawater desalination was started in GCC countries in the 1960s, but significantly desalinated water supply was observed in the past two decades, contributing to the overall water consumption share. Increasing trend of desalination capacities has resulted in higher energy consumption, more than 25% of total energy production in GCC countries. It is estimated that,
with contracted desalination facilities, the total primary energy requirement will grow up to 255.45 GWh in 2025 with 55% increment as compared to 112.47 GWh in 2000 and CO₂ emission will be doubled in 2025 as compared to 60 ktonne/year in 2000 as shown in Table 1. The income loss attributed to desalination was 3.6 billion USD in 2000 and this loss is expected to rise to 31 billion USD in 2025 as calculated by considering yearly fuel price in terms of US$/bbl as shown in Table 1.

<table>
<thead>
<tr>
<th>Year</th>
<th>Specific energy utilization for desalination (GWh_pe)</th>
<th>CO₂ production on the basis of fuel (0.527 tonne/MWh) (ktonne/yr)</th>
<th>Displaced income due to desalination (1 bbl = 1628kWh), (million USD)</th>
</tr>
</thead>
<tbody>
<tr>
<td>1970</td>
<td>6.32</td>
<td>3.3</td>
<td>517.2</td>
</tr>
<tr>
<td>1975</td>
<td>25.19</td>
<td>13.3</td>
<td>2066.0</td>
</tr>
<tr>
<td>1980</td>
<td>62.29</td>
<td>32.8</td>
<td>5107.6</td>
</tr>
<tr>
<td>1985</td>
<td>92.79</td>
<td>48.9</td>
<td>5103.7</td>
</tr>
<tr>
<td>1990</td>
<td>98.26</td>
<td>51.8</td>
<td>3930.6</td>
</tr>
<tr>
<td>1995</td>
<td>109.92</td>
<td>57.9</td>
<td>2748.0</td>
</tr>
<tr>
<td>2000</td>
<td>112.47</td>
<td>59.3</td>
<td>3598.9</td>
</tr>
<tr>
<td>2005</td>
<td>131.93</td>
<td>69.5</td>
<td>7256.2</td>
</tr>
<tr>
<td>2010</td>
<td>163.96</td>
<td>86.4</td>
<td>13116.6</td>
</tr>
<tr>
<td>2015</td>
<td>191.21</td>
<td>100.8</td>
<td>21989.5</td>
</tr>
<tr>
<td>2020</td>
<td>221.97</td>
<td>117.0</td>
<td>26636.7</td>
</tr>
<tr>
<td>2025</td>
<td>255.45</td>
<td>134.6</td>
<td>31164.5</td>
</tr>
</tbody>
</table>

Table 1. Specific energy consumed and CO₂ footprint in GCC countries due to desalination processes

Water production by desalination processes has significant effect on the energy requirement and environment. The intricate nexus between water, energy, and environment has encouraged scientists and engineers to innovate desalination methods with better energy efficiency and environment-friendly processes. Although RO processes are energy-efficient and dominantly used, they have certain limitations with respect to local conditions. For example, the frequent maintenance issues from high operating pressure; water quality problems in term of residuals of boron, chlorides, and bromides; and severe fluctuations in the seawater intake quality are some of the challenges faced by the RO membranes. In the GCC region, frequent occurrence of harmful algae blooms (HABs) that may contain neuroparalytic and diarrhetic toxins which can pass through the pores of membranes lead to health problems. Large fluctuations in the feed water quality have direct implication to the operation and maintenance costs of RO plants [18–20]. Owing to the uncertainty of RO operation, thermal desalination
methods are deemed as the dominant processes employed in desalination market in the Middle East countries, more than 65% of installed capacities.

An innovative solution to strengthen the thermally driven and yet robust multi-effect desalination (MED) is its integration with the heat-driven adsorption desalination (AD) cycle. AD cycle is an emerging low-cost desalination system that requires only low temperature waste heat or solar energy to operate the cycle. The hybridization of both cycles, called MEDAD, extend the range of downstream (last stage) temperature of conventional MED system typically from 40°C to 5°C. The additional number of stages enhances the water production of the MED cycle by 2 - 3 fold at the same top-brine temperatures. In addition, AD integration cycle has the following advantages: (i) it increases inter-stage temperature differential of each MED stages due to the lowering of the bottom-brine temperature; (ii) it helps to scavenge the ambient energy in last part of the MED stages where the latent energy is further recycled; (iii) the AD cycle utilizes only low temperature waste heat; (iv) it has almost no major moving parts; (v) it reduces the chances of corrosion and fouling due to high concentration exposed to low temperature (5°C) in the last stages; (vi) it produces additional cooling effect from last stages of MED operating below ambient temperature; and (vii) significant increase in system performance. The basics of adsorption phenomenon, AD cycle, and its hybrids and desalination processes economics are presented in detail in this chapter.

2. Basic adsorption phenomena

The understanding of an AD cycle is predicated on the availability of basic equilibria–vapor uptake or isotherms of an adsorbent–adsorbate pair at assorted pressures and temperatures. From literature, all isotherms of adsorbent–adsorbate pairs can be categorized into six types (IUPAC) and they are described by many types of empirical and semi-empirical models. The simplest adsorption isotherm model is the classical Langmuir model [21] where it assumes a homogeneous surface with a monolayer vapor uptake where all adsorbent surfaces contain a uniform charged energy. Each pore vacant site is assumed to be filled by a single vapor molecule, forming a single sorption event. Invoking the rate of gas molecules filling the adsorption sites \(\frac{d\theta}{dt}\), as given by Ward and co-workers [22–25], the expression of the Langmuir isotherm model can be derived as follows:

\[
\frac{d\theta}{dt} = K' \left[ \exp \left( \frac{\mu_g - \mu_a}{RT} \right) - \exp \left( \frac{\mu_g - \mu_a}{RT} \right) \right]
\]

(1)

where \(\mu\) is the chemical potential in kJ/mol and subscripts g and a are the gaseous and adsorbed phases, \(T\) is the absolute temperature, and \(K'\) is the dimensionless constant. The isotherm, \(\theta\), can be obtained by integrating the rate equation over the energy level from \(E_c\) to infinity and in this simple case, when \(P \to 0\) then \(\theta \to 0\), and as \(P\) is large, \(\theta\) approaches 1.
A real solid surface of an adsorbent contains geometrical roughness that is formed during its formation or activation process and the energetic heterogeneity has to be accounted for. The gas molecules experienced varying potential at adsorption sites of uneven energy levels and the surfaces are subdivided into infinitesimal pieces. Thus, the total adsorption of a heterogeneous surface is the sum of the product of the sorption event or surface coverage, \( \theta(\varepsilon_i) \) and its probability energy distribution, \( \chi(\varepsilon_i) \), i.e.,

\[
\theta = \sum_i \theta(\varepsilon_i) \cdot \chi(\varepsilon_i) = \int_{-\infty}^{\infty} \chi(\varepsilon) \cdot \hat{\theta}(\varepsilon) \, d\varepsilon
\]

(3)

where the Langmuir model can be applied with a probability function over the entire surface being summed to unity, i.e.,

\[
\int_{-\infty}^{\infty} \chi(\varepsilon) \, d\varepsilon = 1
\]

(4)

Applying a condensation approximation at moderate temperature, the local surface coverage can be simplified to a step-like dirac-delta function as shown in Figure 3, and expressed below;

\[
\lim_{T \to \infty} \hat{\theta}(\varepsilon) = \begin{cases} 0 & \text{for } \varepsilon < \varepsilon_c \\ 1 & \text{for } \varepsilon \geq \varepsilon_c \end{cases}
\]

(5)

where, \( \varepsilon_c \) relates to the adsorption energy in equilibrium conditions. Hence, Equation 3 can be simplified to

\[
\theta = \int_{-\infty}^{\varepsilon_c} 0 \cdot \chi(\varepsilon) \, d\varepsilon + \int_{\varepsilon_c}^{\infty} 1 \cdot \chi(\varepsilon) \, d\varepsilon = \int_{\varepsilon_c}^{\infty} \chi(\varepsilon) \, d\varepsilon
\]

(6)

A solution of \( \chi(\varepsilon) \) to represent the site energy allocation for a real adsorbent surface is obtained by approximating it to a continuous probability distribution function. Depending on the adsorbent surface energy characteristics during adsorption interactions, symmetrical or asymmetrical Gaussian functions are usually assumed, as illustrated in Figure 4.
The details of EDFs, $\chi(\varepsilon)$, for the classic Langmuir–Freundlich [26], Dubinin–Astakhov [27], Dubinin–Raduskevich [28], and Tóth isotherm models are outlined in Table 2, and integrating Figure 3.

Step-like profile yielded from Equation 5 at moderate temperatures

---

Figure 3. Step-like profile yielded from Equation 5 at moderate temperatures

Figure 4. An illustration of symmetrical or asymmetrical Gaussian function to represent the adsorption site energy distribution for classical isotherm models

The details of EDFs, $\chi(\varepsilon)$, for the classic Langmuir–Freundlich [26], Dubinin–Astakhov [27], Dubinin–Raduskevich [28], and Tóth isotherm models are outlined in Table 2, and integrating...
EDFs from the cut-off energy $\varepsilon_c$ to $\infty$, the exact correlation of these isotherm models can be obtained.

<table>
<thead>
<tr>
<th>Model Name</th>
<th>Adsorption Site Energy Distribution $\chi(\varepsilon)$</th>
<th>Isotherm Equation</th>
</tr>
</thead>
<tbody>
<tr>
<td>Langmuir–Freundlich</td>
<td>$\chi(\varepsilon) = \frac{1}{\varepsilon} \exp\left(\frac{\varepsilon - \varepsilon_0}{\varepsilon_c}\right)$</td>
<td>$\beta = \left{\frac{K \exp\left(\frac{\varepsilon_0}{RT}\right)}{1 + \left[K \exp\left(\frac{\varepsilon_0}{RT}\right)\right]^2}\right}^{1/\theta}$</td>
</tr>
<tr>
<td>Dubinin–Astakhov</td>
<td>$\chi(\varepsilon) = \frac{r(\varepsilon - \varepsilon_1)^{r-1}}{E^r} \exp\left(\frac{\varepsilon - \varepsilon_1}{E}\right)$</td>
<td>$\theta = \exp\left(-\frac{RT}{E^r \ln\left(P_0 / P\right)}\right)$</td>
</tr>
<tr>
<td>Dubinin–Radushkevich</td>
<td>$\chi(\varepsilon) = \frac{2(\varepsilon - \varepsilon_1)}{E^2} \exp\left(\frac{\varepsilon - \varepsilon_1}{E}\right)$</td>
<td>$\theta = \exp\left(-\frac{RT}{E^2 \ln\left(P_0 / P\right)}\right)$</td>
</tr>
<tr>
<td>Tóth</td>
<td>$\chi(\varepsilon) = \frac{1}{RT} \left[\exp\left(\frac{\varepsilon - \varepsilon_1}{RT}\right)\right]^{1/\theta}$</td>
<td>$\theta = \left(1 + \left[K \exp\left(\frac{\varepsilon_0}{RT}\right)\right]^2\right)^{1/\theta}$</td>
</tr>
</tbody>
</table>

Table 2. Assorted forms of adsorption site energy distribution functions (EDF) for the Langmuir-Freundlich, Dubinin-Astakhov (DA), Dubinin-Raduskevich (DR) and Tóth isotherm models

2.1. Universal site-Energy probability Distribution Function (EDF)

In the recent development of adsorption isotherm theory, Li [29] proposed a universal model that was able to fit all types of isotherms, as specified in Equation 7. Type I to V patterns at various temperatures could be directly captured by the equation with four regression parameters.

$$ q = \frac{A \phi \exp\left(\beta \frac{P}{P_0}\right) \frac{P}{P_0} + C \frac{P}{P_0}}{1 + \phi \exp\left(\beta \frac{P}{P_0}\right) \frac{P}{P_0}} $$ (7)

and,

$$ \beta = \exp\left(\frac{E}{RT}\right) $$ (8)
where the alphabet $\phi$, $C$ are constants, $t$ is surface heterogeneity factor, and $E_c$ denotes the characteristic energy of the adsorbent–adsorbate pair. These four parameters are required to calculate in the regression process. The rest of the letters have their usual means.

Using the unified adsorption isotherm framework, a universal adsorption site energy distribution function (EDF) was proposed, which relates directly to their isotherm types, and the proposed EDF fitted well with the statistical rate theory of adsorption. The EDF yielded a single peak asymmetrical distribution for Type I which was similar to that for the classical LF, DA, DR, and Tóth isotherm models. The EDF is given as below;

$$A = \left[1 + \phi \exp(\beta)\right] - \frac{C}{\phi \exp(\beta)} \tag{9}$$

$$\chi(\varepsilon) = \frac{\beta^*}{RT} \left[ \phi \exp(\beta^*) \left( \frac{\beta^* + 1}{\beta^*} \right) \right]^{\frac{1}{1-1}} \left[ \frac{A \phi \exp(\beta^*) \left( \frac{\beta^* + 1}{\beta^*} \right) + C}{\phi \exp(\beta^*)} \right]^{\frac{1}{1-1}} \left[ 1 - \phi \exp(\beta^*) \left( \frac{\beta^*}{\beta} \right)^{\frac{1}{1-1}} \right]^{\frac{1}{1-1}} \left[ -C \phi \exp(\beta^*) \left( \frac{\beta^*}{\beta} \right)^{\frac{1}{1-1}} \right]^{\frac{1}{1-1}} \tag{10}$$

where the variable $\beta^*$ is a function of the adsorption site energy $\varepsilon$ and it is expressed as

$$\beta^* = \exp \left( \frac{2E_c - \varepsilon}{RT} \right) \tag{11}$$

From the data available from literature and the proposed Eqs. 9 and 10, the assorted isotherms as categorized by the IUPAC can be successfully captured succinctly by these equations using only four coefficients of regression, and these energy distribution functions and isotherms are depicted in Table 3. For each type of isotherm, the corresponding energy distribution functions (EDFs) have been developed.

The adsorbent–adsorbate interaction is the key in the design of adsorption cycle, which can be functionalized to adopt to warmer ambient temperatures, particularly for operation in the summer period of semi-arid or desert regions. Figures 5(a) and (b) depict two major types of useful adsorbents in water uptake: The former is silica gel Type 3A, suitable for an AD cycle operating below 33°C ambient such as tropical weather conditions. The latter figure depicts the isotherms of zeolite (Z0-alumina phosphate oxide, $\text{Al}_x\text{Ph}_y\text{O}_n\text{H}_2\text{O}$). It has properties that can be tailored for adsorption/desorption at ambient temperatures up to 50°C (corresponding to desorption at 3.8–4.5 kPa (60°C isotherm) and adsorption at 2–2.8 kPa (40°C isotherm).
 thermos-physical properties of both adsorbents are tabulated in Table 4, showing the BET surface-pore areas of 600–800 m$^2$/g.

<table>
<thead>
<tr>
<th>Types of Isotherms (IUPAC Categorization)</th>
<th>Adsorbate–Adsorbent Pair / References</th>
</tr>
</thead>
<tbody>
<tr>
<td>Type I Water–silica gel Type RD</td>
<td>Qiu J. Characterization of silica gel–water vapor adsorption, MEng Thesis, 2004, NUS</td>
</tr>
<tr>
<td>Type I Water–silica gel Type A</td>
<td></td>
</tr>
</tbody>
</table>
Types of Isotherms (IUPAC Categorization)  

<table>
<thead>
<tr>
<th>Type</th>
<th>Adsorbate-Adsorbent Pair / References</th>
</tr>
</thead>
<tbody>
<tr>
<td>II</td>
<td>Water–boehmite</td>
</tr>
<tr>
<td>II</td>
<td>Water–polyvinyl pyrrolidone</td>
</tr>
</tbody>
</table>


Table 3. A summary of the energy distribution functions and isotherms as categorized by the IUPAC

<table>
<thead>
<tr>
<th>Properties</th>
<th>Silica Gel Type 3A</th>
<th>Zeolite FAM Z01</th>
</tr>
</thead>
<tbody>
<tr>
<td>BET surface area [m²/g]</td>
<td>680</td>
<td>147.3</td>
</tr>
<tr>
<td>Porous volume [ml/g]</td>
<td>0.47</td>
<td>0.071</td>
</tr>
<tr>
<td>Apparent density [kg/m³]</td>
<td>770</td>
<td>600–700</td>
</tr>
<tr>
<td>Thermal conductivity [W/m.K]</td>
<td>0.174</td>
<td>0.113 (30°C)</td>
</tr>
<tr>
<td>Heat of adsorption (H₂O) [kJ/kg of H₂O]</td>
<td>2800</td>
<td>3110 (25°C)</td>
</tr>
<tr>
<td>Specific heat capacity [kJ/kg.K]</td>
<td>0.921</td>
<td>(30°C)</td>
</tr>
</tbody>
</table>

Table 4. Thermo-physical properties of the silica gel type 3A and Zeolite (Z01)
3. Design of AD batch-operated cycle

There are five main components of AD system namely: (i) evaporator, (ii) adsorption and desorption reactor beds, (iii) condenser, (iv) pumps, and (v) pretreatment facility. The detailed process diagram is shown in Figure 6. For the batch-operated AD cycle, it involves two main processes.

3.1. Adsorption-assisted-evaporation

In which the vapors generated in AD evaporator are adsorbed on the pore surface area of adsorbent. The heat source is circulated through the tubes of evaporator and seawater is sprayed on the tube bundle. It is observed that the evaporation is initiated by heat source, but during adsorption process the high affinity of water vapor of adsorbent drops the evaporator pressure and contribute in evaporation. The AD evaporator operation temperature can be controlled by heat source temperature that is normally circulated in terms of chilled water. The AD evaporator can operate at a wide range of chilled water temperature varying from 5°C to 30°C to produce the cooling effect as well as at low temperature operation. The vapor adsorption process continues until the adsorbent bed reaches a saturation state.

![Diagram of AD desalination cycle](image)

**Figure 6.** Detailed schematics of an adsorption desalination cycle. The circle (filled) dots are the vapor valves and the pair of triangles refers to the liquid valves. The range of vapor pressures in evaporator and condenser are 1-2 kPa and 5 to 7 kPa.
3.2. Desorption-activated-condensation

In which saturated adsorbent is regenerated using the low-grade industrial waste heat or renewable energy (desorption temperature varies from 55°C to 85°C) and desorbed vapors are condensed in a water-cooled condenser and collected as a distillate water.

It can be seen that two useful effects produced by AD cycle are the cooling effect by the first process “adsorption-assisted-evaporation” and fresh water production by converting the seawater by second process “desorption-activated-condensation”. Useful effects which are cooling and water production can be produced simultaneously by introducing the multi-bed technique.

In multi-bed AD system, the operation and switching technique is used. During operation, one or pair of adsorbent reactor beds undergo the adsorption process and at the same time one or pair of adsorbent reactor beds execute the desorption process. The time for adsorbent reactor beds operation, either adsorption or desorption, depends on the heat source temperature and silica gel quantity packed in a bed. Prior to changing the reactor duties, there is a short time interval called switching in which the adsorber bed is preheated whilst the desorber bed is precooled to enhance the performance of cycle. In AD cycles, the operation (adsorption and desorption) and switching processes are controlled by automated control scheme that can open and close the respective bed hot/cold water valves. During switching operation, all vapor valves are closed so that there is no adsorption or desorption taking place.

4. MED-AD hybrid cycle

MEDAD is a hybrid of two thermal systems namely multi-effect desalination system and adsorption cycle. The main components of this novel thermal hybrid system are (1) multi-effect distillation (MED) system, (2) adsorption desalination (AD) cycle, and (3) auxiliary equipments. In this hybridization system, the last stage of the MED is connected to adsorption beds of AD cycle for the direct vapor communication to adsorption beds. Figure 7 shows detailed flow schematic of MED plant combined with AD system.

Adsorption-based desalination is investigated by many researchers [30–45] and reported that optimal specific daily water production (SDWP) for four bed scheme is about 4.7 kg/kg silica gel. The first adsorption desalination plant was installed in the National University of Singapore (NUS) which consists of four silica gel beds. Ng et al. investigated the processes using chilled water at assorted temperature and demonstrated that the specific water production of the system [46, 47]. They also introduced and patented a novel hybrid desalination method “MEDAD cycle” that is a combination of conventional MED and AD cycle [48, 49]. This novel desalination cycle can mitigate the limitations of conventional MED system to increase the system performance. This combination allows the MED last stage to operate below ambient temperature typically at 5°C as compared to traditional MED at 40°C. This not only reduces the corrosion chances but also increases the distillate production to almost 2 – 3 fold as compared to traditional MED systems.
Figure 7. Detailed P&ID diagram for an experimental MED facility of nominal heating capacity of 12 kW. The first stage comprises a water-fired steam generator whilst the last stage (3rd stage) is open to a water-cooled cooling tower.

4.1. MED-AD simulation

MEDAD cycle simulation have been conducted [50–53] and presented in Table 5. The simulation is based on a fully transient model and the predicted results are compared with conventional MED system. It is observed that at same input parameters such as a top-brine-temperature (TBT), water production can achieve up to two fold increase when the hybridized MEDAD is compared with the MED cycle alone.

<table>
<thead>
<tr>
<th>Equation</th>
<th>Remarks</th>
</tr>
</thead>
<tbody>
<tr>
<td>Modeling equations for steam generator</td>
<td></td>
</tr>
<tr>
<td></td>
<td></td>
</tr>
</tbody>
</table>
\[
\left[ \left( M_{sw} \cdot C_{pw} \right) \right] \frac{dT_{hw}}{dt} = \left( \dot{m}_{in} \cdot c_{p, hw} \right) \cdot \left( T_{sw} \right) - \left( \dot{m}_{in} \cdot h_{fw} \right) \cdot \left( T_{hw} \right) - h_{in} \cdot \Delta v_{in} \cdot \left( T_{hw} - T_{tub} \right) 
\] | Energy balance for the hot water flowing inside the tubes of brine heater. |
| 
\[
\left[ \left( M_{HX,i} \cdot C_{hx,i} \right) \right] \frac{dT_{tub}}{dt} = h_{in,i} \cdot \Delta v_{in} \cdot \left( T_{tub,i} - T_{v,i} \right) - h_{out,i} \cdot \Delta v_{out} \cdot \left( T_{tub,i} - T_{v,i} \right) 
\] | Energy balance for metal tubes. |
| 
\[
\frac{dM_{b,i}}{dt} = \dot{m}_{f,i} - \dot{m}_{v,i} - \dot{m}_{n,i} 
\] | Mass balance for the seawater inventory in the system. |
### Modeling equations for intermediate stages

<table>
<thead>
<tr>
<th>Equation</th>
<th>Remarks</th>
</tr>
</thead>
<tbody>
<tr>
<td>$Q_{in} = \dot{h}<em>{in} \cdot A_i (T_i - T</em>{in}) + (M_{HX,i} \cdot C_{P_{HX,i}}) \frac{dT_{cond,i}}{dt} \cdot \left( \frac{\dot{m}<em>{in} \cdot h</em>{f,\text{cond,i}} + \dot{m}<em>{out} \cdot h</em>{f,\text{cond,tube,i}}}}{\dot{m}<em>{in}} \right) + Q</em>{n,i}$</td>
<td>Energy balance for the condenser side of the steam generator.</td>
</tr>
<tr>
<td>$\frac{dX_{f,i+1}}{dt} = \frac{1}{M_{b,i}} \frac{\dot{m}<em>{f,i} \cdot h</em>{f,i+1} \cdot X_{f,i+1}}{\dot{m}<em>{in}} + \frac{\dot{m}</em>{in}}{M_{b,i}} \cdot X_{f,i+1}$</td>
<td>Material/concentration balance</td>
</tr>
<tr>
<td>$Nu = \frac{\dot{h}<em>{in} \cdot A</em>{in}}{R_{eff,i}} = 0.023 \cdot Re^{0.8} \cdot Pr^{0.4} \cdot d^{0.2} \cdot (\frac{v}{\mu})^{0.8}$</td>
<td>Convective heat transfer coefficient equation</td>
</tr>
<tr>
<td>$R_{eff,i} = -\frac{\ln \left( \frac{d_{out,i}}{d_{in,i}} \right)}{\sum_{i=1}^{n} \frac{C_{p,T}}{d_{in,i}}}$</td>
<td>Tube wall resistance</td>
</tr>
<tr>
<td>$h_{f} \left( \frac{v^{2}}{d_{in}} \right)^{1/3} = 0.0007 \cdot R_{e} \cdot 0.2 \cdot P_{r} \cdot 0.65 \cdot q^{-1/4}$</td>
<td>Falling film evaporation heat transfer coefficient, Han and Fletcher's correlation</td>
</tr>
<tr>
<td>$\frac{1}{U_{i}} = \frac{1}{h_{in}A_{in}} + \frac{1}{R_{eff,i}} + \frac{1}{h_{out,i}A_{out,i}}$</td>
<td>Overall heat transfer coefficient</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>Equation</th>
<th>Remarks</th>
</tr>
</thead>
<tbody>
<tr>
<td>$\left[ M_{HX,i} \cdot C_{P_{HX,i}} \right] \frac{dT_{cond,tube,i}}{dt} \cdot \left[ \frac{\dot{m}<em>{in} \cdot h</em>{f,\text{cond,tube,i}} + \dot{m}<em>{out} \cdot h</em>{f,\text{cond,tube,i}}}{\dot{m}<em>{in}} \right] + Q</em>{n,i}$</td>
<td>Energy balance for the condenser side of the $i^{th}$ effect</td>
</tr>
<tr>
<td>$\left[ M_{HX,i} \cdot C_{P_{HX,i}} \right] \frac{dT_{tube,i}}{dt} \cdot \left[ \frac{\dot{m}<em>{in} \cdot h</em>{f,\text{cond,tube,i}} + \dot{m}<em>{out} \cdot h</em>{f,\text{cond,tube,i}}}{\dot{m}<em>{in}} \right] - h</em>{out,i} \cdot A_{out,i} \cdot \left( T_{cond,tube,i} - T_{tube,i} \right)$</td>
<td>Energy balance for tube metal</td>
</tr>
<tr>
<td>$\frac{dM_{b,i+1}}{dt} = m_{f,i} - \dot{m}<em>{b,i+1} - m</em>{b,i+1}$</td>
<td>Brine inventory balance</td>
</tr>
<tr>
<td>$Q_{n,i+1} = h_{out,i} \cdot A_{out,i} \cdot \left( T_{i+1} - T_{tube,i} \right) \left[ M_{HX,i} \cdot C_{P_{HX,i}} \right] \frac{dT_{tube,i}}{dt}$</td>
<td>Energy balance for evaporator side</td>
</tr>
<tr>
<td>$M_{b,i} \frac{dX_{b,i}}{dt} = \left( \frac{\dot{m}<em>{f,i} \cdot X</em>{f,i}}{\dot{m}<em>{in}} \right) - \left( \frac{\dot{m}</em>{b,i+1} \cdot X_{b,i+1}}{\dot{m}<em>{b,i+1}} \right) - \left( \frac{\dot{m}</em>{b,i+1} \cdot X_{b,i+1}}{\dot{m}_{b,i+1}} \right)$</td>
<td>Material/concentration balance</td>
</tr>
</tbody>
</table>
### Equation

\[ \text{Nu} = \frac{h_{\text{in},i+1}L_{i+1}}{k_{\text{tube},i+1}} = 0.728 \frac{gh_{\text{fg},T_{\text{cond}}}}{\rho_lT_{\text{cond}}(\rho_l - \rho_v)} \left( \frac{T_{\text{cond}}}{K_{l}} \right) \left( \frac{T_{\text{cond}}}{\mu_{l}} \right) \left( \frac{T_{\text{cond}}}{d_i} \right) \left( \frac{T_{v,i+1} - T_{\text{tube},i+1}}{1/4} \right) \]

### Remarks

Nusselt film condensation correlation for the calculation of the heat transfer coefficient inside the condenser tubes

### Equation

\[ U_iA_i = \frac{1}{h_{\text{in},i}A_{\text{in},i}/R_{\text{wall},i}} + \frac{1}{h_{\text{out},i}A_{\text{out},i}} \]

### Remarks

Overall heat transfer coefficient equation

### MED last stage connected with AD beds

\[ \left( \left( M_{\text{v},v}C_p \right) + \left( M_{\text{HX},w}C_p \right) \right) \frac{dT_v}{dt} = \left( m_{f,v}h_{f,v} \right) \frac{dT_f}{dt} - \left( m_{b,v}h_{b,v} \right) \frac{dT_b}{dt} + Q_{\text{in},v} + h_{\text{sat},v} \left( T_{v,v} - T_{\text{sat},v} \right) \]

### Energy balance for the condenser side of the \( i \)th effect

\[ Q_{\text{in},v} = h_{\text{sat},v} \left( T_{v,v} - T_{\text{sat},v} \right) \]

### Table 5. MEDAD modeling equations

Figure 8 shows the transient temperature profiles of a MEDAD cycle. It can be seen that last stages of MED are operating below ambient temperature due to hybridization. It can also be noticed that MED last stages profiles are affected by cyclic AD operation.

![Figure 8. MEDAD cycle components temporal temperature profiles](Image)

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**Figure 8. MEDAD cycle components temporal temperature profiles**
Figure 9 mapped the performance parameters of hybrid MEDAD cycle (concentration, GOR, PR, and WPR). It can be observed that the batched-mode water vapor uptake by the coupled AD cycle dominates the performance of the cycle. The performance of the MEDAD cycle with different additional effects with the conventional eight effect MED cycle as baseline cycle is studied in terms of water production rate.

Figure 9. MEDAD hybrid cycle performance (concentration, GOR, PR and WPR)

Figure 10 shows the quantum jump in the water production rate of the proposed MEDAD cycle. Another aspect of this hybridization is that the desorbed water vapor from the AD cycle can be recycled back into the MED system for further energy recovery.

4.2. MED-AD experimentation

A three-stage MED system is designed [54], fabricated, and installed in NUS as shown in Figure 11. In MED stages, vapor emanation from feed seawater is achieved by falling film-evaporation process. Evaporation energy is recovered by series of reutilization of vapor condensing energy in successive stages of those produced in preceding stages. Process of vapor production and energy recovery by condensation continues until the last stage of MED. The vapors from the last stage are then directed toward AD beds where they are adsorbed on the adsorbent surface. Adsorbent high affinity for water vapor drops the pressure and hence the saturation temper-
The temperature of last stages falls below ambient, typically up to 5°C. It is observed that this drop in pressure and temperature of last stage also affected the operational parameters of the few preceding stages.

**Figure 10.** Percentage improvement in water production rate by the MEDAD cycle

![Percentage improvement in water production rate](image)

**Figure 11.** MEDAD system installed in NUS

![MEDAD system installed in NUS](image)
Experiments are conducted in two steps. In first part, system is operated as a conventional MED at assorted heat source temperature ranges from 38°C to 70°C. In second part, experiments are conducted as a hybrid MEDAD system at assorted heat source temperature ranges from 15°C to 70°C and results are compared with conventional MED system.

Figure 12. MEDAD components temperature profiles at 38°C heat source temperature

Figure 12 shows the instantaneous temperatures of MED and MEDAD components at a heat source temperature of 38°C. It can be seen that steady-state events (minimum temperature fluctuations) occur after 1 hour from start-up and experiments for distillate collection are continued for 4 to 5 hours. It is noticed that the inter-stage temperature difference (ΔT) is more than twice per stage as compared to the conventional MED stages. This is attributed to the vapor uptake by the adsorbent of AD cycle, resulting in the increase of vapor production. The MEDAD cycle yields a stage ΔT from 3°C to 4°C as compared to 1°C or less in the case of MED alone. Table 6 shows the comparison of MEDAD and MED components steady-state temperature values.

Figure 13 shows the distillate production trace at heat source 38°C from MED stages, AD condenser and combined. The batch operated AD production can be seen clearly. At the start of desorption, the production is higher and it drops with time to zero during the switching period while MED stages production is quite stable. Small fluctuations in MED water production may be due to the fluctuations in the spray of the feed that affect the condensation rate. It can be seen that hybridization boost water production 2 – 3 fold as compared to conventional MED system at same TBT. Water production profiles are similar as explained in simulated results.
### Table 6. A comparison of conventional MED and hybrid MEDAD systems components temperatures at different heat source temperatures

<table>
<thead>
<tr>
<th>Heat source temperature (°C)</th>
<th>MED SG</th>
<th>MED Stage-2</th>
<th>MED Stage-3</th>
<th>MEDAD SG</th>
<th>MEDAD Stage-2</th>
<th>MEDAD Stage-3</th>
<th>MEDAD condenser</th>
<th>MEDAD evaporator</th>
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</thead>
<tbody>
<tr>
<td>70</td>
<td>63.9</td>
<td>54.4</td>
<td>62.8</td>
<td>50.8</td>
<td>62.3</td>
<td>46.7</td>
<td>56.6</td>
<td>26.1</td>
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<tr>
<td>65</td>
<td>59.6</td>
<td>50.7</td>
<td>58.7</td>
<td>47.0</td>
<td>58.1</td>
<td>42.5</td>
<td>53.9</td>
<td>25.8</td>
</tr>
<tr>
<td>60</td>
<td>55.4</td>
<td>49.2</td>
<td>54.4</td>
<td>45.7</td>
<td>53.8</td>
<td>41.3</td>
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<td>44.8</td>
<td>49.9</td>
<td>40.8</td>
<td>45.8</td>
<td>25.1</td>
</tr>
<tr>
<td>50</td>
<td>47.4</td>
<td>44.6</td>
<td>46.7</td>
<td>41.6</td>
<td>46.2</td>
<td>37.9</td>
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<td>23.2</td>
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<tr>
<td>45</td>
<td>43.9</td>
<td>41.1</td>
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<td>31.6</td>
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<td>37.4</td>
<td>35.4</td>
<td>37.2</td>
<td>33.5</td>
<td>37.0</td>
<td>30.3</td>
<td>35.3</td>
<td>18.2</td>
</tr>
</tbody>
</table>

Operating limit of Conventional MED. The lower operational points are from MEDAD Hybrids

<table>
<thead>
<tr>
<th>Heat source temperature (°C)</th>
<th>MED AD</th>
<th>MED Stage-2</th>
<th>MED Stage-3</th>
<th>MEDAD AD</th>
<th>MEDAD Stage-2</th>
<th>MEDAD Stage-3</th>
<th>MEDAD condenser</th>
<th>MEDAD evaporator</th>
</tr>
</thead>
<tbody>
<tr>
<td>35</td>
<td>-</td>
<td>30.9</td>
<td>27.9</td>
<td>-</td>
<td>23.5</td>
<td>-</td>
<td>-</td>
<td>16.1</td>
</tr>
<tr>
<td>30</td>
<td>-</td>
<td>26.0</td>
<td>23.7</td>
<td>-</td>
<td>19.1</td>
<td>-</td>
<td>-</td>
<td>12.2</td>
</tr>
<tr>
<td>25</td>
<td>-</td>
<td>22.1</td>
<td>19.9</td>
<td>-</td>
<td>16.3</td>
<td>-</td>
<td>-</td>
<td>10.1</td>
</tr>
<tr>
<td>20</td>
<td>-</td>
<td>18.1</td>
<td>14.2</td>
<td>-</td>
<td>11.3</td>
<td>-</td>
<td>-</td>
<td>7.1</td>
</tr>
<tr>
<td>15</td>
<td>-</td>
<td>13.2</td>
<td>11.5</td>
<td>-</td>
<td>9.4</td>
<td>-</td>
<td>-</td>
<td>5.6</td>
</tr>
</tbody>
</table>

Table 6. A comparison of conventional MED and hybrid MEDAD systems components temperatures at different heat source temperatures

**Figure 13.** Conventional MED and hybrid MEDAD cycle water production profiles at 38°C heat source temperature
Figure 14 shows the comparison of water production of MED and hybrid MEDAD cycles at assorted heat source temperatures. Quantum increase in water production (two- to threefold) can be observed at all heat source temperatures. These results have good agreement with simulation results. It can also be seen that in conventional MED system last stage temperature is limited to 38°C due to condenser operating with cooling water from cooling tower. While in the case of hybrid MEDAD, the last stage temperature can be as low as 5°C because there is no condenser and last stage is connected to AD beds for vapor adsorption. This higher overall operational gap in proposed hybrid MEDAD cycle helps to insert more number of stages (up to 19 stages) as compared to conventional MED system (about 4–6 stages). More number of stages increases the vapor condensation heat recoveries and hence the water production at same top brine temperatures.

5. Exergy analysis for operational cost apportionment

A computation model is developed for the cogeneration plant where the properties of expanding steam, such as enthalpy (h) and entropy (s) at a given temperature and pressure, are computed for the key states in the schematic diagrams of “PP”, “PP+MED,” and “PP+MEDAD” cycles. The approach employed here is to calculate the total exergy changes or destruction across the inlet and exit sections of the equipment. For example, the exergy associated with the turbines is the sum of all contributions from (a) the HP-T unit at the same mass flow rate across it, (b) the LP-T unit until the extraction point, and (c) the exergy changes after the point of steam extraction, i.e., $E_{T,1_2} = E_{T,a} + E_{T,b} + E_{T,c}$. On the basis of total exergy available across whole system, proportion utilized by power plant and desalination system is calculated.
For this comparison study, an assorted range of bled-steam is extracted at low pressure but the mass flow rates are varied to cover the expected practical operational ranges, typically from 10% to 50% of the total flow. At 20% bled-steam from the LP-T, the ratios of power-to-water for exergy and energetic analyses are found to be 95.7%:4.3% (exergy) and 72.2%:27.8% (energetic) methods, respectively. This implies that if the 20% steam were to be continuously consumed by the low pressure turbines (LP-T), its work contribution from LP-T would be insignificant, i.e., a maximum of 4.3%. However, the energetic value of bled-steam accounts for a disproportionate share of 27.8% due primarily to the high latent heat content of the low pressure steam. This ratio of energy-to-exergy shares of the total working steam is found to be 4 – 7 fold higher, descending with the larger amount of bled-steam as shown in Figure 15. The higher effectiveness of working steam incurred at the MED cycle is attributed to the better thermodynamic matching of steam’s latent energy.

![Figure 15. Exergetic and energetic proportions at different percent of steam extraction](image)

On the basis of the above analysis for primary fuel cost and with data from the published literature [55, 56], the life-cycle cost (LCC) of water production is compared for all capital expenditure (Capex) and operation expenditure (Opex), across all proven industrial processes,
as shown in Figure 16. Exergy factor calculated above is utilized only for thermal and electricity cost calculations. Energy based analysis has good agreement with GWI [56] data. It can be seen that by LCC, unit water production cost is highest for PP+MSF which amounts to US$ 1.201/m³, whilst the lowest unit cost is the PP+MEDAD method which is only at US$ 0.485/m³ and this unit cost is even lower than the LCC of reverse osmosis (RO) plants.

6. Summary

Recent developments in adsorption theory, adsorption desalination (AD), and conventional MED desalination cycles have been reviewed in this chapter. We highlight the key role of AD cycles which can be hybridized with the proven cycles such as the MED cycle, exploiting the thermodynamic synergy between the thermally driven cycles that significantly improve the water production yields. Experiments were conducted in a lab-scale pilot MEDAD and confirmed the excellent synergetic effects that boosted the water production up to two- to threefold over the conventional MED. We believe that if the hybrid MEDAD cycles are well optimized and operated, it can achieve high GOR and the projected LCC of water production can be lowered to as low as US$ 0.485/m³.
7. Abbreviation

MED; Multi-effect desalination
AD; Adsorption desalination
RO; Reverse osmosis
SDWP; Specific daily water production
EDF; Energy distribution function
GOR ; Gain output ratio
SG; Steam generator
PR; Performance ratio
WPR; Water production ratio
TBT; Top brine temperature
PP; Power plant
HP-T; High pressure turbine
LP-T; Low pressure turbine
LBT; Lower brine temperature
LCC; Life cycle costing
PDF; Probability distribution function

Symbols

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
</tr>
</thead>
<tbody>
<tr>
<td>$p$</td>
<td>Pressure [kPa]</td>
</tr>
<tr>
<td>$p_0$</td>
<td>Saturation pressure [kPa]</td>
</tr>
<tr>
<td>$R$</td>
<td>Universal gas constant [kJ/kmol.K]</td>
</tr>
<tr>
<td>$T$</td>
<td>Temperature [K]</td>
</tr>
<tr>
<td>$t$</td>
<td>Surface heterogeneity for Tóth [-]</td>
</tr>
</tbody>
</table>

Greek Symbols

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
</tr>
</thead>
<tbody>
<tr>
<td>$\gamma$</td>
<td>Dubinin–Astakhov power factor [-]</td>
</tr>
<tr>
<td>$\varepsilon$</td>
<td>Adsorption site energy [kJ/kmol, or kJ/kg]</td>
</tr>
<tr>
<td>$\varepsilon_0$</td>
<td>Equilibrium adsorption site energy [kJ/kg]</td>
</tr>
<tr>
<td>$\varepsilon_{1,3}$</td>
<td>Reference energy [kJ/kmol]</td>
</tr>
<tr>
<td>$\theta$</td>
<td>Adsorption uptake [kg/kg of adsorbent]</td>
</tr>
</tbody>
</table>
θ
Local adsorption uptake [kg/kg of adsorbent]

μ
Chemical potential [kJ/kmol]

χ(ε)
Energy distribution function [-]

Subscript

α
Adsorbed phase

g
Gaseous phase

Acknowledgements

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References


[8] Hoekstra AY, Mekonnen MM. Global water scarcity: the monthly blue water footprint compared to blue water availability for the world’s major river basins, Value of water research report series no. 53, UNESCO-IHE Institute for Water Education.


[16] Al-Zubari WK. The Water-Energy Nexus in the GCC Countries Evolution and Related Policies, Sixth “Zayed Seminar” on Green Economy: Success Stories from the GCC, 8–9 May 2013, UN virtual water learning center, the Arabian regional center, Arabian Gulf University, Bahrain.


[19] Huttner KR. Overview of existing water and energy policies in the MENA region and potential policy approaches to overcome the existing barriers to desalination using renewable energies. Desalin Water Treat 2013;51:87–94.


[29] Li A. Experimental and theoretical studies on the heat transfer enhancement of adsorbent coated heat exchangers, Doctor of Philosophy, Department of Mechanical Engineering, National University of Singapore, 2014.


[34] Ng KC. Recent developments in heat-driven silica gel-water adsorption chillers. Heat Transfer Eng 2003;24:1–3.


